

5.4 Design Of Eccentric Shear And Moment Resisting Connections Girders

Example 4

Design a hand operated overhead crane, which is provided in a shed, whose details are

Capacity of crane = 50kN

Longitudinal spacing of column = 6m

Center to center distance of gantry girder = 12m

wheel spacing = 3m

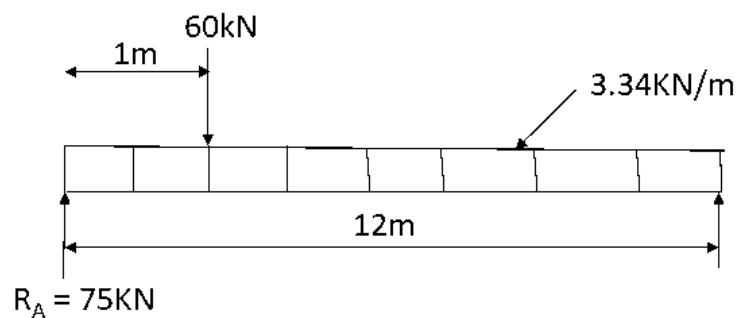
Edge distance = 1m

Weight of crane girder = 40kN

Weight of trolley car = 10kN

Solution:

Step 1 Find wheel load



To find support reaction,

Weight of crane girder per meter span

$$= 40/12$$

$$= 3.34 \text{ kN/m}$$

Weight of crane and trolley added together and placed at 1m

$$= 50 + 10$$

$$= 60 \text{ kN}$$

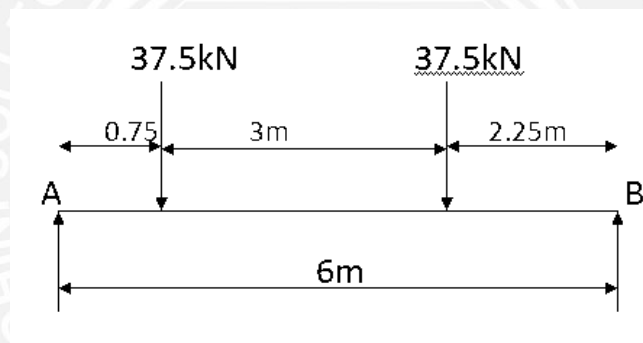
$$R_A \times 12 = 60 \times 11 + 3.34 \times 12^2 / 2$$

$$R_A = 75 \text{ kN}$$

$$\text{Wheel load} = R_A / 2$$

$$= 37.5 \text{ kN}$$

Step 2 Find max BM in gantry girder



$$R_A = 46.88 \text{ kN}$$

$$R_B = 28.12 \text{ kN}$$

$$\text{Max BM} = 28.12 \times 2.25$$

$$= 63.27 \text{ kNm (at trailing wheel)}$$

Adding 10% for impact,

$$M_1 = 1.1 \times 63.27$$

$$= 69.60 \text{ kNm}$$

Max BM due to self weight of girder and rail taking total weight as 1.2 kN/m

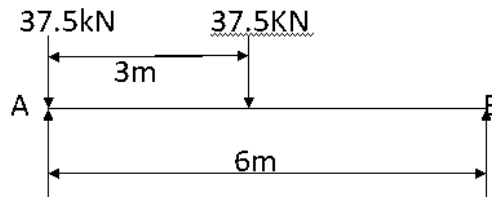
$$M_2 = w l^2 / 8$$

$$= 1.2 \times 6^2 / 8$$

$$= 5.4 \text{ kNm}$$

$$\text{Total BM, } M = 75\text{KNm}$$

Step 3 Find max shear



$$\begin{aligned} S_F &= R_A \\ &= (37.5 \times 6 + 37.5 \times 3) / 6 \\ &= 56.25\text{KN} \end{aligned}$$

Step 4 Find lateral loads

25% of lateral load/num of wheel

$$\begin{aligned} &= 0.025 \times 60 / 2 \\ &= 0.75\text{KN} \end{aligned}$$

Max BM due to lateral load ,

$$\begin{aligned} \text{ML} &= (36.37 / 37.5) \times 0.75 \\ &= 1.27\text{KNm} \end{aligned}$$

Step 5 Selection of section

Economic depth of section

$$\begin{aligned} &= L / 15 \\ &= 6000 / 15 \\ &= 400\text{mm} \end{aligned}$$

Let us try ISMB 450@710.2N/m

IS Code 800;2007, pg138,

$$\text{Flange criteria, } b/t_f = 75/17.4$$

$$= 4.31 < 9.4$$

$$\text{web criteria, } d/t_w = 415.4/9.4$$

$$= 44.19 < 83.9$$

∴ section is plastic

Step 6 Shear capacity

$$F_{vd} = f_y \times A / \sqrt{3} \times \gamma_{mo}$$

$$= 250 \times 450 \times 9.4 / \sqrt{3} \times 1.10$$

$$= 555043 \text{ N}$$

$$= 555 \text{ kN}$$

$$F_v / F_{vd} = 56.25 \times 1.5 / 555$$

$$= 0.152 < 0.6$$

Check for torsional buckling,

$$t_f / t_w \leq 17.4 / 9.4 = 1.85 < 2$$

$$\beta_{LT} = 1.2 \text{ for plastic section}$$

$$M_{cr} = \text{elastic critical moment}$$

$$M_{cr} = \beta_{LT} \pi^2 EI_y / (KL)^2 \left[1 + 1/20 \left[(KL/r_y) / (h/t_f) \right]^2 \right]^{0.5}$$

$$KL = 1.0 \times 6000$$

$$= 6000 \text{ mm}$$

$$M_{cr} = 1.2 \pi^2 \times 2 \times 10^5 \times 834 \times 10^4 \times 450 /$$

$$2 \times 6000^2 \left\{ 1 + 1/20 \left[(6000/30.1) / (450/17.4) \right]^2 \right\}^{0.5}$$

$$= 246 \times 10^6 \text{ Nmm}$$

Factored longitudinal moment,

$$M_f = 75 \times 1.5$$

$$= 112.5 \text{ kNm}$$

Factored lateral moment ,

$$M_{fL} = 1.27 \times 1.5$$

$$= 1.91 \text{ kNm}$$

Lateral BM capacity,

$$M_{dL} = Z_{py} \cdot f_y / 1.10$$

$$= [Z_{ey} / 2 \times \text{shape factor} \times f_y] / 1.10$$

$$= [(111.2 \times 10^3) / 2 \times 1.15 \times 250] / 1.10$$

$$= 14.53 \times 10^6 \text{ Nmm}$$

$$= 14.53 \text{ kNm}$$

For safety,

$$M_f / M_{dL} [\text{longitudinal}] + M_f / M_{dL} [\text{lateral}] \leq 1.0$$

$$= 112.5 / 174.34 + 1.91 / 14.53$$

$$= 0.78 < 1.0$$

Hence, section selected is adequate and safe.